

Subject:- Structural Dynamics

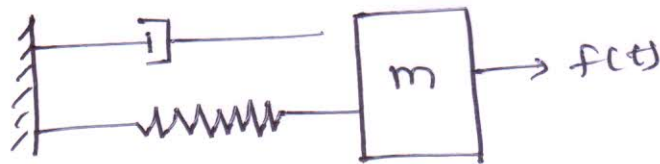
Subject code:- CVSE103

Q. NO. 1 (a) Ans:-

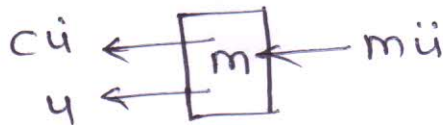
Derivation for response of a damped SDOF system for free vibration.

i.e. $f(t) = 0$.

SDOF system model



SDOF system spring mass model



The equation of motion for damped free vibration

$$m\ddot{u} + c\dot{u} + ku = 0 \quad \text{--- (1)}$$

dividing by m to above eqn

$$\ddot{u} + \frac{c}{m}\dot{u} + \frac{k}{m}u = 0$$

$D^2 + \frac{c}{m}D + \frac{k}{m} = 0$, Roots of the eqn are

$$D = \frac{-\frac{c}{m} \pm \sqrt{\left(\frac{c}{m}\right)^2 - 4\frac{k}{m}}}{2}$$

$$= -\frac{c}{2m} \pm \sqrt{\left(\frac{c}{2m}\right)^2 - \frac{k}{m}}$$

$$\frac{c}{2m} = \gamma \omega$$

$$\therefore c = 2m\eta\omega$$

$$\therefore D = -\eta\omega \pm \omega\sqrt{\eta^2 - 1} = -\omega\eta \pm i\omega\sqrt{1-\eta^2}$$

$$\omega_d = \omega\sqrt{1-\eta^2}$$

\(\therefore\) The solution of above eqn (1) is

$$\boxed{y(t) = C_1 e^{(-\omega\eta + i\omega_d)t} + C_2 e^{(-\omega\eta - i\omega_d)t}} \quad \text{--- (2)}$$

$$= e^{-\eta\omega t} [(C_1 \cos\omega_d t + C_1 i \sin\omega_d t) + [C_2 \cos\omega_d t - i C_2 \sin\omega_d t]]$$

$$\therefore e^{i\theta} = \cos\theta + i \sin\theta$$

$$e^{-i\theta} = \cos\theta - i \sin\theta$$

$$\therefore y(t) = e^{-\eta\omega t} [A \cos\omega_d t + B \sin\omega_d t]$$

where, $A = C_1 + C_2$ & $B = C_1 - C_2$

The constants are calculated based on Boundary conditions.

$$y(t) = e^{-\eta\omega t} [A \cos\omega_d t + B \sin\omega_d t] \quad \text{--- (3)}$$

Applying Boundary conditions, At, $t=0, y_0=0$ & $\dot{y}_0=V_0$

$$y_0 = A, \text{ from eqn 3}$$

$$\dot{y}(t) = V_0 = -\omega\eta [A \cos\omega_d t + B \sin\omega_d t] e^{-\eta\omega t} + e^{-\eta\omega t} [-A \omega_d \sin\omega_d t + B \omega_d \cos\omega_d t]$$

$$\therefore V_0 = -\omega\eta A + B \omega_d \Rightarrow V_0 = -\omega\eta y_0 + B \omega_d$$

$$\therefore B = \frac{V_0 - \omega\eta y_0}{\omega_d}$$

Sub. value to A & B in above eqn 3

$$u(t) = \left[y_0 \cos \omega_d t + \left(\frac{v_0 - y_0 \omega_n}{\omega_d} \right) \sin \omega_d t \right] e^{-\gamma \omega t}$$

$u(t)$ = total Response

y_0 = initial displacement

v_0 = initial velocity

Q. No. 1 (b) :- Ans :- Consider the ~~impulse~~ impulse of the force $F(z)$ as shown in fig. below at time t during the interval dz

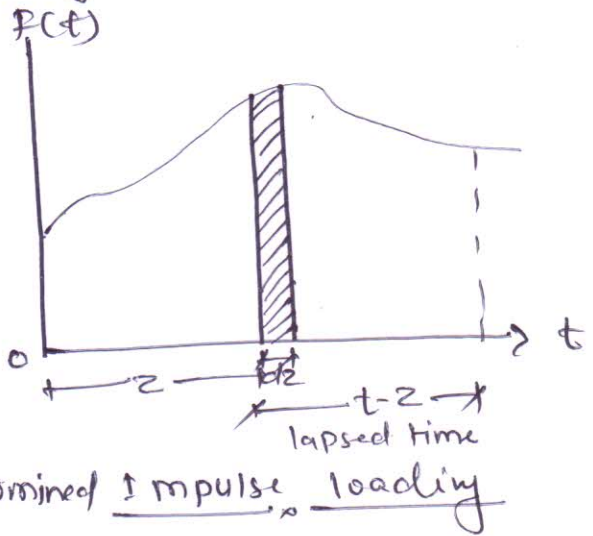
which represented by the shaded Area.

The impulse is equal

to $F(z) dz$ and it is acting on a body of mass

m , produces a change in velocity which can be determined

from Newton's Second law of motion.



$$F = ma = m \cdot \frac{du}{dt}$$

After rearranging

$$du = \frac{F(z) \cdot (dz)}{m} \quad \text{--- (a)}$$

Here, $F(z) dz$ is impulse & du is the change in velocity during the interval dz i.e. change in velocity during the interval dz .

This incremental velocity may be considered to be an initial velocity of the mass at time t . This impulse $F(z) dz$ acting on the system represented by the undamped oscillator.

$$y(t) = y_c(t) + y_p(t) \quad \text{--- (1)}$$

i.e. complementary function $y_c(t) = A \sin \omega_n t + B \cos \omega_n t$

$$y_c(t) = A \sin \omega_n t + B \cos \omega_n t \quad \&$$

particular integral $y_p(t)$ can be obtained by Duhamel's integral.

The change in velocity as initial velocity \dot{y}_0 together with initial displacement $y_0 = 0$ at time τ , producing a displacement at a later time t , is given by.

$$y_c(t) = A \sin \omega_n t + B \cos \omega_n t \quad \text{--- (2)}$$

using initial condition,

$$t = 0, y_0 = 0 \quad \& \quad \dot{y}_0 = \dot{y} = \dot{y}_0$$

sub. in eqn (2) we get

$$B = 0 \quad \& \quad A = \frac{\dot{y}_0}{\omega_n} = \frac{\dot{y}_0}{\omega_n}$$

$$\therefore y_c(t) = \frac{\dot{y}_0}{\omega_n} \sin \omega_n t \quad \text{--- (3)}$$

But the duration (2) represented is actually incremental time $(t - \tau)$

$$\therefore y_c(t) = \frac{\dot{y}_0}{\omega_n} \sin \omega_n (t - \tau)$$

$$= \frac{F(\tau) (d\tau)}{m \omega_n} \sin \omega_n (t - \tau) \quad \text{--- (4)}$$

The loading may be represented as a series of short impulses at successive incremental times $d\tau$. Thus each short impulse producing its own response at time t of the form given in above eqn (4). Therefore the total response created by all the incremental impulses is found by integrating eqn (4) from $\tau = 0$ to time t .

$$u(t) = \frac{1}{m\omega_n} \int_0^t F(z) \sin \omega_n (t-z) \cdot dz$$

This is ~~now~~ called as Duhamel's integral which represents the total displacement produced by the exciting force $F(z)$ acting on the undamped oscillator.

i) Total displacement by Duhamel's integral subjected to a loading, i.e. $F(t) = F_0$, for undamped.

$$m\ddot{u} + ku = F_0$$

$$\ddot{u} + \frac{ku}{m} = F_0/m$$

$$\therefore u(t) = \frac{1}{\omega_n m} \int_0^t F_0 \sin \omega_n (t-z) \cdot dz$$

$$= \frac{F_0}{\omega_n m} \int_0^t \sin \omega_n (t-z) dz$$

$$= \frac{F_0}{\omega_n m} \left[\frac{-\cos \omega_n (t-z)}{-\omega_n} \right]_0^t$$

$$= \frac{F_0}{\omega_n^2 m} [1 - \cos \omega_n t]$$

$$\boxed{u(t) = \frac{F_0}{k} [1 - \cos \omega_n t]}$$

ii) Total Displacement, when $F(t) = F_0 \cdot t$

$$\therefore u(t) = \frac{1}{\omega_n m} \int_0^t F_0 \cdot z \sin \omega_n (t-z) \cdot dz$$

$$= \frac{F_0}{\omega_n m} \int_0^t z \sin \omega_n (t-z) \cdot dz$$

$$= \frac{F_0}{\omega_n m} \left\{ \left[\frac{z \cos \omega_n (t-z)}{\omega_n} \right]_0^t - \int_0^t \frac{\cos \omega_n (t-z)}{\omega_n} dz \right\}$$

